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Title: OUTER RING OF A WHEEL BEARING, : Dated: December 31, 2008
AND AXIAL SECURING ELEMENT
FOR SAID OUTER RING

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CERTIFICATE OF ACCURACY

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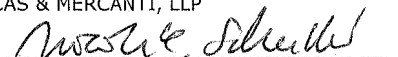
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BY:


Nicole K. Schuller

I, Nicole Schuller, hereby declare as follows:

1. I am conversant in both English and German.

2. The attached is an accurate English version of the German Patent Application

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Nicole Schuller

Title of the Invention

Outer ring of a wheel bearing and axial securing element for said outer ring

Description

Field of the Invention

The invention relates to an outer ring of a wheel bearing having a radial flange, the flange extending radially outward at the axial end of the outer ring of tubular configuration, and to an axial securing means of the outer ring on a wheel carrier.

Background of the Invention

DE 39 40 395 A1 describes customary arrangements of wheel bearings. Wheel bearings are either accommodated completely in a wheel carrier or the like, or are suspended on a flange. Tests have shown that the wheel modules with wheel bearings which are as a rule suspended on the wheel carrier via a flange on the outer ring make up a higher weight proportion of the unsprung masses than wheel bearings which are arranged in a hole of a wheel carrier. The construction of the wheel bearings, which are suspended on the wheel carrier and are therefore at least partially self-supporting, and their suspension is to be of particularly rigid design, which results in the increased weight proportion in comparison with the wheel bearings which are accommodated in holes.

DE 39 40 395 A1 thus describes a wheel bearing having an outer ring, rolling bodies and two inner rings, in which the outer ring has a radial flange axially between the end

sides and away from the end sides, with which radial flange the wheel bearing is suspended. The wheel bearing is fixed axially and radially on the wheel carrier via the flange. A hollow cylindrical section of the outer ring which adjoins the flange axially is seated in a hole of the carrier, with the result that the outer ring bears against the carrier axially by means of the flange and at least partially radially at least with the section. The weight proportion of the solidly configured flange on the outer ring is relatively high. This has a disadvantageous effect on the overall weight of the unsprung masses of a wheel module on the vehicle.

Furthermore, DE 39 40 395 A1 describes a wheel bearing which is accommodated completely in a hole of a wheel carrier. The outer ring is of solid configuration and manufactured by material removing machining. A press fit which is customary in the roller bearing field secures the outer ring in the hole. In the case of loading of the wheel bearing during driving operation, the outer ring tends to move axially despite the press fit. To counteract this, the outer ring is secured in the hole with a securing ring axially on one side and is supported on an inner shoulder on the other side. The preparation of a bearing seat of this type is complicated, as the shoulder makes material removing machining difficult in the axial direction. Furthermore, the notch is to be introduced for the securing ring. The press fit also often becomes loose during driving operation, with the result that notches are produced on the carrier or on the outer ring by wear, and/or the known and undesirable creaking noises occur, as a result of the micromovements of the outer ring with respect to the carrier. The weight of the solid outer ring has a disadvantageous effect on the total balance of the unsprung masses on the vehicle.

Summary of the Invention

It is therefore the object of the invention to provide an outer ring for a wheel bearing unit, with the installation of which into the wheel bearing unit the above-mentioned disadvantages are avoided.

This object is achieved in accordance with the subject matter of claim 1.

Instead of through holes, the flange has recesses which pass axially through the flange and are open radially to the outside, for fastening the flange to a wheel carrier. The weight proportion of the overall weight of the flange is therefore reduced considerably, as the material of an annular section of the flange which usually encloses the recesses on the outside is omitted.

One embodiment of the invention provides for the recesses to be of arcuate design. As an alternative, however, recesses are also provided which extend radially from the outside in the manner of a slot in the direction of the rotational axis of the wheel bearing.

During the manufacture of a flange of this type, the flange which is initially provided with through holes is, for example, machined from the outside to remove material by machining until the superfluous proportion of the material which delimits the holes radially on the outside is removed from the flange. As an alternative to this, the contour according to the invention of the flange is produced by punching. This is particularly advantageous when the outer ring is cold formed. Machining to remove material from the edge is therefore superfluous. In this case, the recesses and the

other sections of the flange which are accessible radially from the outside have a punched edge from a punching process for cutting at least the recesses.

The flange is formed, axially between the end sides of the flange and at the same time axially spaced apart from the end sides, on the outer ring or fastened to the outer ring. In so doing, the wheel bearing is suspended by means of the flange on the wheel carrier or the like or is supported radially at least partially in the wheel carrier.

As an alternative, one embodiment of the invention provides for the section of the outer ring, on which raceways for the rows are formed at least partially, to be supported radially in the wheel carrier. As a result, the entire wheel bearing is as a rule surrounded by the wheel carrier. The flange is formed at the axial end of the outer ring and protrudes radially outward at one end of the hole. The flange either bears directly axially against the wheel carrier or is supported axially on the wheel carrier via spacer means. The wheel bearing is secured axially with respect to the carrier in the hole via the flange, as the flange is fastened to the wheel carrier with suitable means. Axial movement of the outer ring during driving operation is avoided. Moreover, the channel at the transition from the flange to the outer ring is relieved, as the outer ring is supported radially in the wheel carrier under load.

In this context, wheel carriers are to be understood as all connecting constructions for wheel bearing modules and wheel bearing units of driven and nondriven wheels, to/in which the wheel bearing modules or wheel bearing units are fastened. The wheel carriers which are cast or forged or manufactured in another suitable way are manufactured from all conceivable materials, that is to say from steel and/or from aluminum alloys at least around the hole for accommodating the wheel bearing. A

wheel bearing according to the invention which is fixed axially on the wheel carrier by means of the flange of the outer ring is fixed axially also when the radial seat of the wheel bearing has become loose radially, for example, in a carrier made from an aluminum alloy on account of influence factors mentioned at the beginning.

In this context, wheel bearings are to be understood as all one row, two row and multiple row wheel bearings having rollers and/or balls as rolling bodies, having one or more inner rings of one part or multiple parts, which are configured in a very wide range of arrangements, such as in a radial ball bearing arrangement or angular contact ball bearing arrangement or as in a radial roller bearing arrangement or angular contact roller bearing arrangement. As an alternative, one or all of the inner raceways is/are not formed on one/a plurality of inner rings but is/are formed directly on a hub which is arranged in the wheel bearing.

Before the installation of the wheel bearing in the wheel carrier, the hub of a wheel bearing module for a driven or nondriven wheel is fixed in the wheel bearing, for example by a flange rim on the bearing. Here, the flange rim reaches behind the bearing arrangement in such a way that the hub is fixed on the wheel bearing such that it cannot be separated from the wheel bearing. As an alternative, the hub or a bolt, with which the wheel is mounted in the wheel bearing such that it can rotate with respect to the wheel carrier, is inserted only when the wheel bearing is fixed to the carrier.

The wheel bearing for mounting driven and/or nondriven wheels on vehicles is installed with the outer ring in the wheel carrier in such a way that the flange of the outer ring points toward the vehicle or away from the vehicle. On wheel bearings, with

which driven wheels are mounted, alternatively the flange of the outer ring either bears against that side of the wheel carrier which faces the wheel flange, or the flange bears against a side of the wheel carrier which faces away from the wheel flange.

One embodiment of the invention provides for the outer ring to be relatively thin-walled and therefore elastic in comparison with the rigid housing which surrounds the hole. The wall of the outer ring has thin cross sections. The outer ring, in particular a ring which is manufactured by cold forming, is partially hardened or is hardened overall; or an outer ring which is manufactured as desired without material removing processing after the hardening, is therefore unstable in itself or unround as an individual part on the outside and, in particular, on the inside at the raceways, in such a way that the optimum geometry which is necessary for rolling contact is not realized initially. This particularly relates to the diameter and the roundness of the section which is considered transversely with respect to the rotational axis, with which section the outer ring is pressed into the hole, before mounting in the wheel carrier. In this case, optimum geometry is to be understood as the roundness of the raceways in rolling contact, which ensures optimum rolling conditions in the bearing. Shape accuracies of this type are usually produced by turning and grinding on solid bearing rings which are machined with removal of material and the housing holes for accommodating the bearing rings. The manufacture of geometry of this type is time consuming and expensive. However, the invention provides for the finished contour of the outer ring to have, at least at the section, the above-mentioned deviations from the optimum geometry. The hole of the carrier corresponds to the optimum requirements on account of its geometry which is produced by removing material.

The deviations of the outer contour of the outer ring from the optimum geometry are

compensated for by the elastic, partially plastic, behavior of the outer ring when the outer ring is pressed into the hole. To this end, the outer ring has, initially at least at the section, an external diameter which is greater than the internal diameter of the hole. The excess dimension is also required for the press fit, with which the outer ring is seated in the carrier. The outer ring is constricted radially inward by being pressed into the hole, and is adapted to the roundness of the hole. The shape accuracies of the hole and also the excess dimension are transmitted almost completely to the geometry of the raceways, as the housing of the carrier is rigid in comparison with the outer ring. The hole in the wheel carrier which is comparatively rigid with respect to the outer ring has substantially its initial geometry before the wheel bearing is pressed in, when the wheel bearing is seated in the hole. The functionally required roundnesses on the raceway are therefore only produced on the outer ring after the outer ring is pressed into the hole. Moreover, the elastic change in shape is also advantageous, inter alia, for the necessary freedom from play of the wheel bearing when the wheel bearing is mounted completely as a unit.

The thin walled outer ring is preferably configured in one piece with the flange, but the flange can also be fastened to the outer ring by welding or by a force transmitting and/or form fitting seat. One embodiment of the invention provides for the outer ring to be made from formed material. Accordingly, the shape of the outer ring with all its shaped elements is produced by forming. Cutting or material removing machining is restricted only to a very small amount of the machining in comparison with the amount of chipless machining. Only excess material, edges, burrs and the like are therefore removed from the shaped part by cutting or punching. Only the raceways are optionally machined in a material removing manner by precision machining such as grinding, lapping or polishing. Cold forming is to be understood as all the forming

processes, in which the contour of the hollow outer ring can be manufactured by stretching or upset forging, expansion or contraction, and the shape of the starting material can be changed plastically in the process without material being cut. Processes of this type are, for example, drawing, deep drawing, rolling, pressing and combinations of the above-mentioned processes.

Tubes and metal sheets for example, are provided as blanks for the manufacture of the outer rings. A blank comprising a tube is machined to form the finished outer ring by expansion, rolling, contracting, upset forging and the folding over of edges. Outer rings which are manufactured from a metal sheet are manufactured by drawing and further individual processes mentioned above or combinations of the latter. In this case, one embodiment of the invention provides for the flange to be exactly as wide, axially from the end side of the outer ring as far as the wheel carrier against which the flange bears, as the starting material of the metal sheet was thick before the outer ring was manufactured. Preferred materials are cold formable bearing materials, such as 100Cr6 or else all suitable deep drawing steels.

The outer ring is distinguished by a low weight, as starting material having a small wall thickness or small sheet thickness has been used. The thickness of the starting material preferably lies in the range from 2.4 to 5 mm. The overall proportion of the wheel bearing unit in the weight of the unsprung masses is reduced by way of the thin walled outer ring which is manufactured by removing material or without cutting.

The flange is preferably fastened to the wheel carrier with bolts which, for example, are screwed or pressed into the wheel carrier. Other fastening elements are conceivable, such as clamps or the like which engage behind the flange at at least one

fastening edge on a side of the flange which faces away axially from the wheel carrier. As an alternative to this, bolts are provided which are fastened to the wheel carrier by welding or screwing and onto which in each case a nut is screwed and prestressed axially against the flange. The bolts reach through axial recesses on the flange. Here, the head of each of the bolts bears axially at the same time against the wheel carrier and against the flange. The uniform contact of the heads is secured by axial faces on the wheel carrier and on the flange of the outer ring which lie in a common plane. To this end, the wheel carrier has projections which protrude axially from the end-side contact face, on which the flange bears axially against the wheel carrier. The projections protrude axially to such an extent that the axial face which is formed on the end of the projections lies axially in one plane with the axial face on the flange which faces away from the carrier.

As an alternative to the axial projections, the wheel carrier is provided with an annular section which is circumferential or interrupted in the circumferential direction. The annular section surrounds the flange radially on the outside. The heads of the fastening means bear jointly against this axial annular section.

As viewed in a longitudinal section through the outer ring along the rotational axis of the outer ring, the concave contour of the channel between the flange and the outer ring merges at a first transition, for example, into a circularly annular face of the flange and at the second transition, for example, into a cylindrical circumferential face of the section. As an alternative, the first and the second transitions end, for example, in each case in an annular groove, which is shaped in the manner of an undercut, in the flange or in the section. Here, it is particularly significant that a perpendicular spacing between an imaginary axial extension of the circumferential surface of the

flange and the first transition is smaller than a spacing which is parallel to the rotational axis between an imaginary radial extension of the circumferential surface and the second transition.

Here, the channel is advantageously described by at least two radii which follow one another between the transitions, whereas at least a first radius merging radially into the flange at the transition and at least a second radius merging axially into the first section at the transition. In longitudinal section in the direction of the transitions, the radii are either separated from one another by a straight line, or, preferably, merge into one another between the flange and the section.

The outer ring is supported in the wheel carrier at least in sections at least radially with respect to the rotational axis in such a way, that the flange which is formed axially on the end side of the outer ring bears axially against the wheel carrier and radially against the outer ring, without the carrier being in contact with the channel. Additional stresses in the channel as a result of the influence of sharp edges of the carrier are also avoided by a chamfer of sufficient dimensions on the hole.

Further embodiments of the invention and embodiments are described in greater detail in figures 1 to 12. They show in detail:

Detailed Description of the Drawings

Figure 1 shows a wheel bearing module 1 in a longitudinal section along a rotational axis 11. The wheel bearing module is provided with an outer ring 2, with two rows of rolling bodies 3, with an inner ring 4, with a hub 5 and with an articulation bell 6. The

structural unit of the wheel bearing module which is preassembled from the individual parts 2, 3, 4, 5, 6 is seated in a wheel carrier 7.

The wheel bearing 8 comprising the outer ring 2, the rolling bodies 3, in this example in the form of balls, cages 9, seals 10, the inner ring 4 and the hub 5 is preassembled so as to hold together. To this end, the rolling bodies 3 and the inner ring 4 are held in the outer ring 2 by means of the hub 5. Here, one row of the rolling bodies 3 is supported on a raceway 2a of the outer ring 2 and on a raceway 5a which is formed directly on the hub 5. The other row of the rolling bodies 3 is arranged between a further raceway 2a and a raceway 4a of the inner ring 4. The wheel bearing 8 is prestressed without play by means of a flange rim 5b on the hub 5 via the inner ring 4 and the outer ring 2, at least when the wheel bearing 8 is mounted in the wheel carrier 7. The hub 5 is fixed nonreleasably to the wheel carrier 8 via the flange rim 5b.

The hub 5 is configured to be substantially rotationally symmetrical about the rotational axis 11 and has an axial through hole 5c. A stub 6a on the articulation bell 6 reaches through the through hole 5c axially. The articulation bell 6 is secured axially on the hub 5 via a nut 12. A wheel flange 5d which points radially to the outside is formed on the hub 5. A connection which is fixed rotationally about the rotational axis 11 is produced between the articulation bell 6 and the wheel flange 5d via tooth profiles 13 on the stub 6a and on the hub 5.

Axial holes 5e are formed on the wheel flange 5d, into which axial holes 5e wheel bolts (not shown) engage for fastening a vehicle wheel. Furthermore, the wheel flange 5d has axial recesses 5f which lie axially opposite a flange 2c at least once per revolution of the wheel flange 5d about the rotational axis 11, in such a way that the wheel

flange 5d does not cover the fastening element 14 axially on account of the recesses 5f. The diameter D_1 of the recess 5f which is configured as an axial through hole 15 is greater than the greatest radial dimension A_1 of the head 14a.

The outer ring 2 is formed from two hollow cylindrical sections 2b and from the radial flange 2c. The wheel bearing 8 is supported radially in a hole 7a of the wheel carrier 7 via the sections 2b. At one of the sections 2b, the outer ring 2 merges radially into the flange 2c via a channel 2d. The flange 2c bears axially on the outside against the wheel carrier 7 and is engaged from behind by the fastening elements 14 in the form of bolts on the side 2e which faces axially away from the wheel carrier 7. The fastening element 14 is prestressed axially fixedly against the flange 2c with the head 14a, as a shaft 14b which is fixed on the bolt is fixed axially in a fastening hole 7b of the wheel carrier 7. The fastening elements 14 are optionally pressed or screwed into the fastening holes 7b.

Figure 2 shows a complete view of the outer ring 2. The outer ring 2 is a component which is manufactured by cold forming, the flange 2c of which has radially protruding sections 2g. The recesses 2f are made in three of the sections 2g as axial through holes 16.

Figure 2a shows the shape deviations of the outer ring 2 as an individual component in comparison with the internal geometry 2r of the hole 7a. On account of its thin wall after cold forming, the outer ring 2 has an external geometry 2r, the diameter D_A of which is initially greater by the excess dimension $2 \times U$ than the internal diameter D_I of the hole 7a. In addition, the external geometry 2s deviates by the amount $2 \times V$ from the internal geometry 2r of the fastening hole 7b. After mounting of the outer

ring 2 in the fastening hole 7b, the outer ring 2 is constricted to such an extent that the external geometry 2s corresponds substantially to the diameter D_I and the internal geometry 2r.

The outer ring 2 has a radial shoulder 2h. The radial shoulder 2h is arranged between the raceways 2a, the raceways 2a being formed at least partially on the radial shoulder 2h. An annular groove 2k is made in the outer ring 2 radially from the outside. On the one hand, the annular groove 2k is resulting from the displacement of material out of the outer ring 2 for forming the radial shoulder 2h, and on the other hand, a certain elasticity is imparted to the outer ring 2 in the region of the raceways 2a.

Figure 2b, an enlarged illustration of the detail Z from figure 1, shows the channel 2d in an extremely enlarged illustration which is not to scale. The channel 2d is of concave configuration and merges into a circular cylindrical circumferential surface 2m of the section 2b at a first transition 2l. The channel 2d merges at the transition 2n into a circularly annular face 2p at the flange 2c. The perpendicular spacing S between the imaginary axial extension of the circumferential surface 2m and the transition 2n is smaller than a spacing X which is parallel to the rotational axis 11 between an imaginary radial extension of the circularly annular face 2p and the transition 2l.

Figure 2b also shows that the contour of the channel 2d deviates from a contour 2q which is shown with dash dotted lines and is described by a radius r in a longitudinal section through the outer ring 2. The channel 2d is described in the longitudinal section of the radii r_1 and r_2 . The radius r_1 merges into the flange 2 at the transition 2n and the radius r_2 merges axially into the section 2b at the transition 2l. The radii r_1

and r_2 merge into one another between the flange 2c and the section 2b.

The wheel carrier 7 bears axially against the flange 2c and radially against the section 2b in such a way that the wheel carrier 7 and the channel 2d are spaced apart from one another at least as far as the transitions 2l and 2n. The maximum stresses radially below the recesses 2f are approximately a third higher at the contour 2q which is described by the radius r than the stresses in a channel 2c which is described by the radii r_1 and r_2 .

The wheel flange 5d lies axially opposite the flange 2c and protrudes radially to a greater extent than the flange 2c (figure 1), with the result that the flange 2c is initially covered by the wheel flange 5d initially axially in the direction of the wheel carrier 7.

Figure 3 and figure 4 show the wheel bearing module 1 in different front views. Figure 3 shows the wheel bearing module 1, without articulation bell and without wheel carrier, along the line III-III in the direction according to figure 1 which is marked with the arrows. Figure 4 shows the wheel bearing module 1 in a view which is opposite to that of figure 3, with a view of the wheel flange 5d in the direction of the wheel carrier 7. It can be seen, in particular, from figure 4 that the wheel flange 5d covers the flange 2c on the outer ring 2 completely axially in the direction of the wheel carrier, but in each case one of the recesses 5f, at the same time as the other recesses 5f, lies axially opposite on the wheel flange 5d once per revolution of the wheel flange 5d about the rotational axis 11, in such a way that the wheel flange 5d does not cover the flange 2c axially at the recesses 2f and at the edges of the recesses 2f.

The term "the edges of the recess" is to be understood as the region which adjoins the recesses and on which the fastening elements bear against the flange. For this reason, the recesses 5f are designed to be at least large enough that it is possible for the fastening means 14 with head and also a tool which surrounds the fastening means 14 at the head 14a to be guided axially through the recesses 5f.

The recesses 5f are distributed around the rotational axis 11 with the same pitch T_1 with respect to one another, the pitch T_1 being predefined by the angle β , in this case $\beta = 72^\circ$.

The holes 5e in the wheel flange 5d are provided in each case with an internal thread 5g (fig. 1), into which the wheel bolts are screwed. It is conceivable to use holes 5e without an internal thread at the same time for axial access to the holes 16. This is possible, as an outer envelope circle which surrounds all holes 15 on the outside is greater than an outer envelope circle which surrounds all holes 16 on the outside, and an envelope circle which bears against all holes 15 on the inside is smaller than an inner envelope circle which bears against all holes 16.

The number of recesses 5f on the wheel flange 5 (in this case, five) is greater than the number of recesses (in this case, three) 2f on the flange 2c. However, the number of recesses can also coincide.

As can be seen from figure 4, each of the surplus recesses 5f in one of the sections 2g without holes 16 lies axially opposite at the same time as the other recesses 5f. These two sections 2g, which are without the recess 2f, as can be seen from figure 3 in particular, are provided for the contact of axial supports for the flange 2 when being

pressed into the hole 7a of the wheel carrier. Here, the axial supports reach axially through two of the recesses 5f.

Figure 5 shows a wheel bearing 17 with a wheel flange 18. Radially protruding sections 18a of the flange have axial holes 18b, into which wheel bolts (not shown) engage for fastening a vehicle wheel. Between every two circumferentially adjacent sections 18a, a recess according to the invention 18c, open to the outside is formed on the wheel flange. Instead of axial through holes, the flange 19a has arcuate recesses 2f which are open radially to the outside. The wheel flange 18 is rotated into position about the rotational axis 11 with respect to the flange 19a in such a way, that the recesses 2f are not covered by the wheel flange 18 in the axial direction.

Figure 6 shows a wheel bearing 20 for a nondriven vehicle wheel, in which optionally different wheel flanges can be mounted, with an outer ring 19 which is manufactured without cutting. In figure 8, the outer ring 19 is shown as an individual component in an overall view and has the radial flange 19a with the recesses 2f of arcuate configuration. The wheel bearing 20 has two of the inner rings 4, on which in each case one of the raceways 4a is formed.

Figure 7 shows a wheel flange 21, in which the recesses 5f which are formed as holes 15 lie radially closer to the rotational axis 11 than the holes 5e for accommodating the wheel bolts. Both the holes 15 and the holes 5e are distributed on the circumferential side with the same pitch with respect to the rotational axis 11.

Figures 9 and 10 show a wheel bearing 23 in a wheel carrier 24. Figure 9 shows a frontal view of the wheel bearing 23 in the wheel carrier 24 without a wheel flange.

Figure 10 shows a longitudinal section through the unit along the line X-X from figure 9. The wheel bearing 23 has the outer ring 22, the rolling bodies 3, cages 9 and two of the inner rings 4. The outer ring 22 is seated in a radially supported manner in a through hole 24a of the wheel carrier. A chamfer 24b adjoins the through hole 24a on the wheel side of the wheel carrier 24. The chamfer 24b is of sufficient dimensions, with the result that a channel 19b at the transition from the outer ring 19 to the flange 19a is not touched by the wheel carrier 24 (figure 10).

The wheel carrier 24 has a planar face 25, against which the flange 19a bears axially. Projections 25a protrude axially from the planar face. The projections 25a adjoin the flange 19a radially at those locations where the flange 19a has the axial recesses 2f. The axial projections 25a have axial cutouts 25b which are configured arcuately and merge into fastening holes 24c in the wheel carrier 24. In each case one fastening element 14 is seated fixedly in one of the fastening holes 24c of the wheel carrier and is in each case reached around by in each case one recess 2f and the cutout 25b of the respective projection 25a. Each of the center axes 24d of the fastening holes 24c extends radially between a recess 2f and a cutout 25b, in the same direction as the rotational axis 11. The head 14a of the fastening element 14 bears axially against axial faces 25c and 19c. Here, the axial face which is formed on the flange 19a and the axial face 25c which is formed on the projection 25a lie in a common plane E1.

One embodiment of the invention provides for the axial face 25c to initially protrude approximately 0.05 mm to 0.2 mm axially beyond the axial face 25c, before the fastening elements 14 are introduced into the fastening holes 24c. During insertion and fastening of the fastening elements 14 in the fastening holes 24c, for example by screwing in, the flange 19a is axially deformed elastically or plastically to such an

extent that the axial faces 25c and 19c lie jointly in the plane E1. Permanent axial securing of the outer ring 19 on the wheel carrier 24 is produced.

Figures 11 and 12 show the wheel bearing 23 in a wheel carrier 24 of alternative design to the illustrations according to figures 9 and 10. Figure 11 shows a frontal view of the wheel bearing 23 in the wheel carrier 24 without a wheel flange. Figure 12 shows a longitudinal section through the unit along the line XII-XII from figure 11.

The wheel carrier 24 has a planar face 25, against which the flange 19a bears axially. An annular section 27 protrudes axially beyond the planar face 25. The annular section 27 reaches around the flange 19a and has axial cutouts 27a which adjoin the recesses 2f radially, are of arcuate design and merge into fastening holes 24c in the wheel carrier 24. In each case one fastening element 14 is seated fixedly in one of the fastening holes 24c of the wheel carrier and is reached around by in each case one recess 2f and the cutout 27a. The head 14a of each fastening element 14 bears axially against an axial face 27b of the annular section 27 and at the same time against the axial face 19c of the flange 19a. Here, the axial face 19a which is formed on the flange 19a and the axial face 27b lie in the common plane E1.

List of Designations

1	Wheel bearing module
2	Outer ring
2a	Raceway
2b	Section
2c	Flange
2d	Channel
2e	Side
2f	Recess
2g	Section
2h	Radial shoulder
2k	Annular groove
2l	Transition
2m	Circumferential surface
2n	Transition
2p	Face
2q	Contour
2r	Internal geometry
2s	External geometry
3	Rolling body
4	Inner ring
4a	Raceway
5	Hub
5a	Raceway
5b	Flange rim

5c	Through hole
5d	Wheel flange
5e	Holes
5f	Recess
6	Articulation bell
6a	Stub
7	Wheel carrier
7a	Hole
7b	Fastening hole
8	Wheel bearing
9	Cage
10	Seal
11	Rotational axis
12	Nut
13	Tooth profile
14	Fastening element
14a	Head
14b	Shaft
15	Hole
16	Hole
17	Wheel bearing
18	Wheel flange
18a	Section
18b	Hole
19	Outer ring
19a	Flange

19b	Channel
19c	Axial face
20	Wheel bearing
21	Wheel flange
22	Outer ring
22a	Flange
22b	Circular annular face
23	Wheel bearing
24	Wheel carrier
24a	Through hole
24b	Chamfer
24c	Fastening hole
24d	Center axis
25	Planar face
25a	Projection
25b	Cutout
25c	Axial face
27	Annular section
27a	Cutout
27b	Axial face

Patent claims

1. An outer ring (2, 19) of a wheel bearing (8, 23) having a radial flange (2c, 19a), the flange (2c, 19a) extending radially outward at the axial end of the outer ring (8, 19) of tubular configuration, and the flange (2c, 19a) having recesses (2f) which pass axially through the flange (2c, 19a) and are open radially to the outside, for fastening the flange (2c, 19a) to a wheel carrier (7, 24).
2. The outer ring as claimed in claim 1, in which the recesses (2f) are of arcuate design.
3. The outer ring as claimed in claim 1, the outer ring (2, 19) being cold formed.
4. The outer ring as claimed in claim 1 on a wheel bearing (8, 23), the outer ring (2) being supported radially at least partially in a wheel carrier (7, 24) and the flange (2c) being fixed axially here to the wheel carrier (7, 24) by way of fastening elements (14), the flange (2c) being engaged from behind by the fastening elements (14) on a side (2e) of the flange (2c) which faces axially away from the wheel carrier (7, 24) and, as a result, being held axially on the wheel carrier (7, 24) axially.
5. The outer ring as claimed in claim 5, in which bolts reach through the recesses (2f), the fastening elements (14) being the bolts.
6. The outer ring as claimed in claim 1, in which the fastening elements (14) are heads (14) of the bolts.

7. The outer ring as claimed in claim 1, in which the flange (2c) bears axially against the wheel carrier (7, 24) at least in sections.

8. An axial securing means of an outer ring (2) of a wheel bearing (23) on a wheel carrier (24), in which the outer ring (2) bears axially against the wheel carrier (24) with a radial flange (2c) and the flange (2c) is fixed axially to the wheel carrier (24) by way of fastening elements (14), the flange (2c) being engaged from behind by the fastening elements (14) on a side (2e) of the flange (2c) which faces axially away from the wheel carrier (24) and, as a result, being held axially on the wheel carrier (7, 24) axially, and each of the fastening elements (14) at the same time bearing axially against the wheel carrier (24) and against the flange (2c).

9. The axial securing means as claimed in claim 8, in which the fastening elements (14) are bolts with heads (14a), each of the bolts being fixed in the wheel carrier (24) and engaging from behind the flange (2c) with a head (14a) on that side on the flange (2c) which faces away from the wheel carrier (24), and the head (14a) bearing at the same time both axially against the flange (2c) and against the wheel carrier (24).

10. The axial securing means as claimed in claim 9, in which each of the heads (14a) bears axially against an axial projection (25a) of the wheel carrier (7), the projections (25a) adjoining the flange (2c) radially.

11. The axial securing means as claimed in claim 9, in which the heads (14a) bear against a common axial annular section (27) of the carrier (24), the annular section (27) surrounding the flange (2c) circumferentially.